

Liquid sloshing and impact in a closed container with high filling

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1 Introduction

Violent liquid sloshing is of concern to cargo tank designers due to the problems of safety in extreme loadings. It is an interesting topic with still some disputable problems. For example Abramson, Bass, Faltinsen and Olsen [1] investigated the sloshing and resultant loads in liquid natural gas carriers for different tank geometries and liquid fill depths. Ibrahim [3] provided a comprehensive study with examples of free tank motions. Cooker [2] analysed with experiments a horizontal rectangular wave tank which swings at the lower end of a pendulum. Ten, Malenica and Korobkin [5], presented a semi-analytical approach for fluid-structure interactions inside tanks in different impact situations with high and low fillings. Our work is about the sloshing of standing waves inside a closed highly filled container. Unsteady two-dimensional (2-D), irrotational flow is treated. The liquid-roof interaction is discussed with and without the effect of gravity and a comparison is made. The short-time model of the liquid-roof impact is governed by a Mixed Boundary Value Problem (MBVP), which is solved numerically and using the asymptotic methods.

2 Mathematical formulation

In figure 1, a stationary highly filled rectangular tank, containing an inviscid incompressible liquid is shown. The flow is 2-D and irrotational and surface tension force is neglected. The container has height H and length $2L$ and lies in the region $\tilde{y} \geq 0$. Here $\tilde{y} = 0$ is the bottom and $\tilde{y} = H$ is the roof of the container, $\tilde{y} = H - h$ is the still water level, and $\tilde{x} = \pm L$ are the rigid impermeable walls. The lengths, time, velocity potential and pressure are scaled by H , $\sqrt{H/g}$, $h\sqrt{Hg}$ and ρgh , respectively, where g is the gravitational acceleration and ρ is the constant density. The small parameter $\epsilon = h/H \ll 1$ is responsible for linearisation. In non-dimensional variables (without tilde), the initial shape of the free surface is given by the equation $y = f(x)$, $f(x) = 1 - \epsilon + \frac{2\epsilon}{\lambda} \sum_{n=1}^{\infty} a_n \cos(k_n x)$, a_n and k_n are known constants and $\lambda = \frac{L}{H}$.

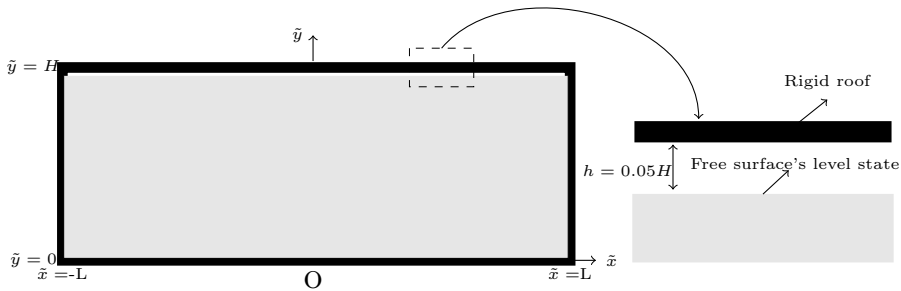


Figure 1: Container description with free surface at its level state.

Considering the container to be without the rigid roof, in non-dimensional linearised initial bound-

The combined kinematic and dynamic boundary conditions (with gravity included) lead to a system of the form

$$\vec{\eta} = \vec{\eta}_l - A^{-1}B\vec{P} - A^{-1} \sum_{m=1}^{M-1} G^{t_m} \vec{p}^{t_m}, \quad (4)$$

where the initial pressure \vec{p}^{t_1} , found in the previous section, with respect to this algorithm is given. The tri-diagonal matrix A is associated with unknown and known surface elevation vectors, $\vec{\eta}$ and $\vec{\eta}_l$ respectively. The matrix G is calculated at every time step while the matrix B is independent of time and depends only on the time step length. The pressure \vec{P} and surface elevation $\vec{\eta}$ are to be determined at the instant $t = t_M$. However, the fact that on the free surface, $x_{c0}(t_M) < x < 1$, we have $p(x, t_M) = 0$ and on the impact region, $0 < x < x_{c0}(t_M)$, we have $\eta(x, t_M) = 1$, with some rearrangements makes the system (4) solvable on its own. The free surface and the wetted region are distinguished and updated at each time step by calculating the position of $x = x_{c0}(t_M)$ as part of the solution. The pressure vector \vec{p}^{t_m} is known from the previous time steps $t = t_m$, for $1 \leq m \leq M-1$.

Continuing with the stretched variables introduced from Section 3, we study the gravity influence on the length of the wetted region during the impact stage, that is $x_{c0}(t) + \delta \cdot x_{c1}(t)$. The correction due to gravity $x_{c1}(t)$ in Figure 4b, is found to be almost completely insignificant at the early stage of impact. As time goes on the effect of gravity is that it decreases the length of the wetted region and even at the very late period of this stage, its effect is found to be small. Figure 4a shows the wetted length with and without gravity. More results of the numerical simulation will be presented at the workshop.

5 Acknowledgements

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References

- [1] H. N. Abramson, R. Bass, O. Faltinsen, and H. Olsen. Liquid slosh in LNG carriers. In *Symposium on Naval Hydrodynamics, 10th, Proceedings, Pap and Discuss, Cambridge, Mass, June 24-28, 1974.*, number Proceeding, 1976.
- [2] M. J. Cooker. Water waves in a suspended container. *Wave Motion*, 20(4):385–395, 1994.
- [3] R. A. Ibrahim. *Liquid sloshing dynamics: theory and applications*. Cambridge University Press, 2005.
- [4] A. Korobkin. Formulation of penetration problem as a variational inequality. *Din. Sploshnoi Sredy*, 58:73–79, 1982.
- [5] I. Ten, Š. Malenica, and A. Korobkin. Semi-analytical models of hydroelastic sloshing impact in tanks of liquefied natural gas vessels. *Philosophical Transactions of the Royal Society A: Mathematical, Physical and Engineering Sciences*, 369(1947):2920–2941, 2011.
- [6] H. Wagner. Phenomena associated with impacts and sliding on liquid surfaces. *Z. Angew. Math. Mech*, 12(4):193–215, 1932.

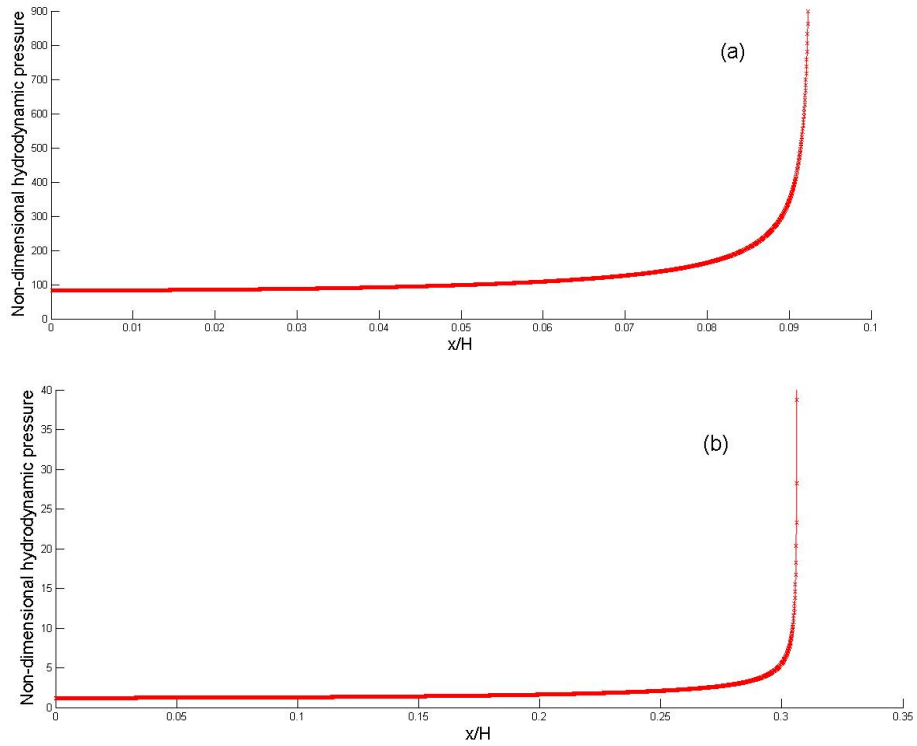


Figure 3: Semi-analytical pressure distribution without gravity at: (a) $t = 1.0790$; (b) $t = 1.5760$.

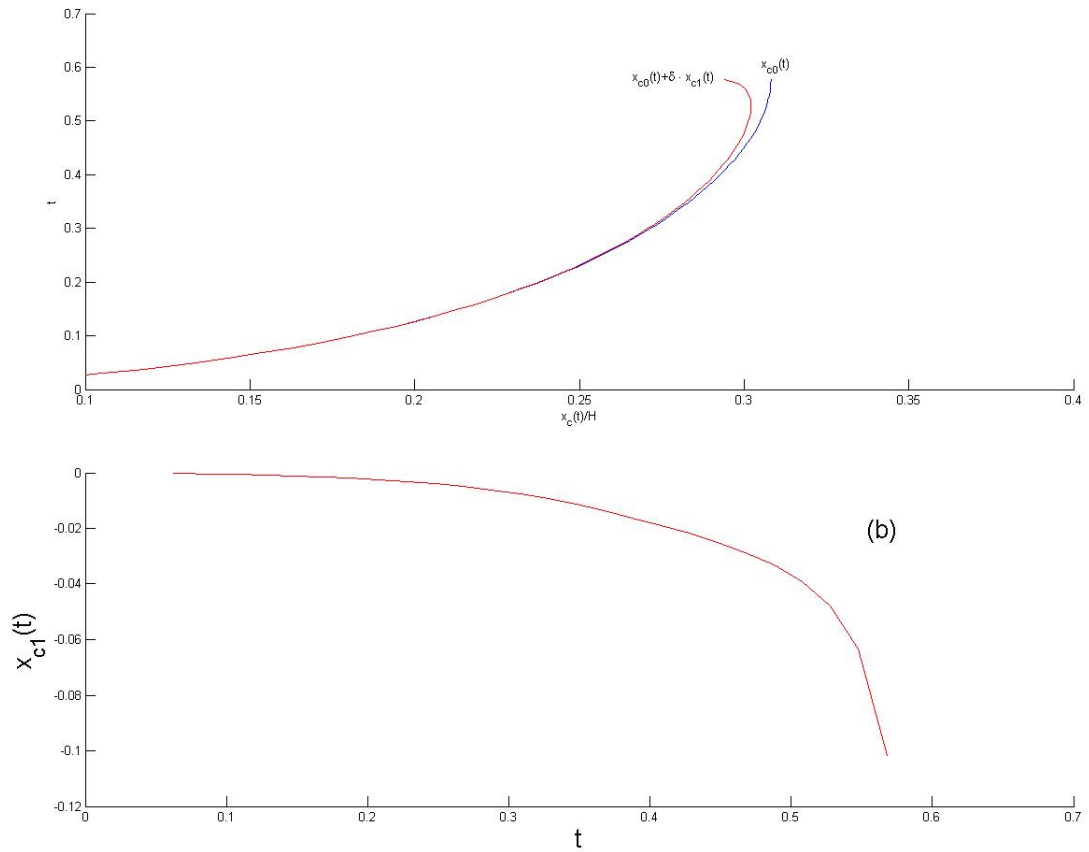


Figure 4: Contact point position with correction due to gravity, here $t = 0$ is the impact time: (a) The leading order contact point, blue line, and the contact point with correction due to gravity, red line; (b) details of correction $x_{c1}(t)$ due to gravity, for the contact point position.